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# Experimental investigation on low-velocity impact response of wood skinned sandwich composites with different core configurations



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<i>Keywords:</i> Wood skin Sandwich structures Impact behaviour Damage	In this paper, an experimental investigation on the low-velocity impact response of wood skinned hybrid sandwich composites was presented. Several alternative design configurations were developed by using rubber- cork and E-glass composite layers between the foam core and wood skin in order to improve the impact per- formance of conventional sandwich composites. Low-velocity impact (LVI) testing was performed using a drop weight test machine at different impact energies and destructive cross-sectioning was performed to examine the interior damage growth and penetration depth of the specimens. The impact performance of the specimens was evaluated in terms of energy absorption capacity, maximum contact force and penetration depth. The multi-core design concept significantly improved the energy absorption capacity with a reduced extent of impact induced damage. The proportion of recyclable materials in each configuration and the energy absorption level per unit

cost were also presented for the interest of product designers.

#### 1. Introduction

Recyclable and sustainable materials in structural applications are becoming increasingly popular due to the strict environmental and legislative restrictions. Although fibre reinforced plastic (FRP) sheet materials have been extensively used to develop advanced sandwich structures in order to maximise the specific bending rigidity and strength, they are relatively unsustainable as a result of difficulties in recycling processes. Thus, renewable materials like woods are emerging as a possible alternative to non-renewable materials [1-3]. For instance, wood-based eco-structures are widely used for meeting the requirements of marine, automotive and environmental-friendly building applications [4,5]. For marine applications as an example, wood based sandwich structures are mainly used as components of the decks, partitions or elements for ceilings and floor structures [6]. In addition to conventional sandwich configurations, an effective combination of different foam, wood and cork materials have been used as skin and core materials, aiming at superior properties such as low density, good thermal and acoustic insulation [7-12].

During service life, sandwich structures experience low velocity impact loadings such as tool drop during manufacturing or maintenance, moorings or floating objects, ice floes and debris [13]. It is generally accepted that low-velocity impacts (LVI) occur at velocities below 10 m/s, and can reduce the strength of whole structure under quasi-static and dynamic loads due to the localised internal skin and/or core damage [14,15].

It was shown that the increased core density and the skin thickness improves the LVI damage resistance of foam core sandwich composites [16–19]. As an alternative, hybrid sandwich structure concepts have been introduced in order to increase the LVI resistance with intermediate layers strategically placed between the face sheets and the core material [20]. It was observed that introducing ductile intermediate layers beneath external skins in foam core sandwiches improved protection against core crushing under LVI loadings [21–24]. On the other hand, using a stiffer composite sheet into a traditional single sandwich structure decreased the local effect of the impact energy and so it was spread in a wider area within the structure [25,26]. Experimental results indicated that an increased damage resistance may be obtained as a function of the mechanical properties of the intermediate layers and their location [21–26].

Eco-friendly structures receive increasing attention due to their environmental benefits such as recyclability and sustainability. Despite their popularity, they have intrinsically lower mechanical properties which have to be complemented with synthetic materials to form stronger hybrid structures. Wood materials are relatively brittle and absorb lower specific energy (J/kgm<sup>2</sup>) under LVI loadings [13,27]. For instance, balsa wood cores perform well under static loading, but they fail catastrophically under impact loading due to the low fracture

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toughness along the grain direction [28]. With regard to plywood structures, at high impact energy levels, they are perforated with heavy loss of structural integrity, which can be undesirable in most applications [29]. Previous studies also revealed that the impact damage tolerance of such wood-based materials can be increased by using them in conjunction with composite skins in sandwich designs [29-31]. Shin et al. [30] showed that the energy absorption of wood-based sandwich structures was comparable with the aluminium honeycomb sandwich structures. Susainathan et al. [29] observed that glass fibre-skinned sandwich structures with plywood cores exhibited comparable energy absorption capabilities and lower indentation in comparison to reference material, which made of Nomex honevcomb core and carbon or glass reinforced skins. It was stated that the recyclability of these new designs is questionable. In our earlier work [31], ashwood was found to be stronger against the impact loading in comparison to pinewood when used as intermediate layers in classical sandwich structures.

Despite the wealth of research efforts on the LVI response of conventional sandwich panels, novel structural sandwich designs require detailed experimental characterisation under given sets of design constraints. Motivated by the current trend towards the bio-composites, new sandwich composites containing synthetic PVC foam core with natural-based layers were manufactured in the present work. By employing a compressible intermediate layer and a stiffer composite internal sheet, it was aimed to enhance the LVI performance of woodskinned sandwich composites. The effects of rubber-cork layer thickness and foam core material on the impact resistance and damage mechanisms of the wood-skinned sandwich composites were characterised by load-displacement curves, energy absorption rates, contact forces and penetration depths. The characteristics of the impact damage mechanisms were analysed by the cross-sectional images of the impacted specimens.

#### 2. Experimental study

#### 2.1. Materials and manufacturing method

Heat-treated ashwood (thermowood) was chosen as face sheets (skin) whereas rubber cork was used as the intermediate layer between the ashwood skin and the foam core to absorb the LVI energy and minimise the excessive local foam crushing under the impact point. Rubber cork layers are generally applied for flooring applications due to their good impact absorption capability, 100% recyclability and ease of maintenance [32–34].

The wood skinned sandwich composite panels were manufactured by the hand lay-up technique. 5 mm thick heat-treated ashwood (Thermo Wood<sup>®</sup>) material with a density of 650 kg/m<sup>3</sup> was used as the face sheet material for all panels whereas low and high density PVC foams were used as core materials which were denoted as LPVC and HPVC, respectively. Table 1 summarizes the physical and mechanical properties of the PVC foams obtained from the manufacturer's datasheet [35]. The effective mechanical properties of the ashwood material were measured according to the DIN standards and are given in Table 2.

Five specimens were tested for both compressive and bending tests. The ashwood specimens with dimensions of  $25 \text{ mm} \times 50 \text{ mm} \times 500 \text{ mm}$  (thickness × width × span length) were tested under three-point

bending according to DIN EN 310 [36] standard. The dimensions of the compression test specimens were  $60 \text{ mm} \times 20 \text{ mm} \times 20 \text{ mm}$  (parallel to grain direction) according to DIN EN 52,185 [37] standard.

Several design combinations of sandwich structures were developed with different foam thicknesses and alternating foam type locations with respect to the impacted face in addition to rubber-cork sheets which were placed between the ashwood skins and the foam cores. The properties of the rubber-cork material are given in Table 3 [38]. Fig. 1(a) shows the baseline wood skinned single foam sandwich (SFS) design where only the 25 mm and 15 mm thick HPVC foams were sandwiched between the ashwood skins. The SFS specimens with rubber-cork intermediate lavers under the ashwood skins were introduced as the second design as shown in Fig. 1(b) and were referred to as SFSR specimens. In the third design, wood skinned multi-foam layered sandwich (abbreviated as MFSR) concepts were formed by introducing a 1 mm thick internal composite sheet between two PVC foam core layers, see Fig. 1(c). The internal composite sheet was made of [0/90]<sub>s</sub> E-glass biaxial non-crimp fabric reinforcement with an areal density of  $600 \text{ g/m}^2$  and vinyl ester resin matrix material (see Table 4). The in-plane mechanical properties of internal sheet were determined with coupon tests following the ISO and ASTM test standards [39-41]. The PVC foams together with the internal composite sheet were considered as a separate unit and produced by a wet layup process while these foam core units were joined to the ashwood skins and rubber-cork layers by using a polyurethane solvent-based one component adhesive. After a total curing time of 24 h, the sandwich panels were cut into  $100 \text{ mm} \times 100 \text{ mm}$  square specimens for impact tests. The manufacturing matrix of eight different sandwich specimen designs is listed in Table 5.

#### 2.2. Drop weight impact test

The sandwich specimens were subjected to low-velocity impact test by using Instron CEAST Fractovis Plus-7526 drop weight testing machine. It consists of a drop-weight tower, an impactor, a velocity sensor, a data acquisition system and an anti-rebound system to prevent multiple impacts on the specimen for an individual test. Impact loading was applied by a hemispherical steel impactor of 12.7 mm tip diameter and 5 kg total mass at the centre of the specimens, which were clamped by a pneumatic fixture with a 76.2 mm hole diameter as shown in Fig. 2. In accordance with the ASTM D3763 standard [42], the clamped sandwich specimens were impacted with impact energy levels ranging from 15 J to 75 J at room temperature. The test velocities for 15, 30, 45, and 75 J impact energy levels were 2.468, 3.49, 4.275, and 5.518 m/s, respectively. Three specimens were tested for each sandwich configuration and average values were calculated. In order to obtain the impact response of the specimens and to reach quantitative conclusions, various parameters such as contact force, impactor displacement, and absorbed energy values were recorded by a data acquisition system during the impact event which are detailed in the following sections.

#### 3. Experimental results and discussion

A load-displacement curve in an impact test contains significant information about the damage process [43]. The displacement term in

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Properties	of	PVC	foam	cores	[35]	
riopenues	UI.	1 1 0	roam	COLCS	55	•

Trademark	Density (kg/m <sup>3</sup> )	Compressive Strength (MPa)	Compressive Modulus (MPa)	Tensile Strength (MPa)	Tensile Modulus (MPa)	Shear Strength (MPa)	Shear Modulus (MPa)	Color
Airex C70.55	60	0,9	69	1,3	45	0,85	22	Yellow
Airex C70.75	80	1,45	104	2	66	1,2	30	Green
	Trademark Airex C70.55 Airex C70.75	TrademarkDensity (kg/m³)Airex60C70.55	TrademarkDensity (kg/m³)Compressive Strength (MPa)Airex600,9C70.55	TrademarkDensity (kg/m³)Compressive Strength (MPa)Compressive Modulus (MPa)Airex600,969C70.55	TrademarkDensity (kg/m³)Compressive Strength (MPa)Compressive Modulus (MPa)Tensile Strength (MPa)Airex600,9691,3C70.551,451042Airex801,451042	TrademarkDensity (kg/m³)Compressive Strength (MPa)Compressive Modulus (MPa)Tensile Strength (MPa)Tensile Modulus (MPa)Airex600,9691,345C70.551,45104266	TrademarkDensity (kg/m³)Compressive Strength (MPa)Compressive Modulus (MPa)Tensile 	TrademarkDensity (kg/m³)Compressive Strength (MPa)Compressive Modulus (MPa)Tensile StrengthTensile Modulus (MPa)Shear Strength (MPa)Shear Modulus (MPa)Shear Modulus (MPa)Shear Modulus (MPa)Shear Modulus (MPa)Shear Modulus (MPa)Shear Modulus (MPa)Shear Modulus (MPa)Shear Modulus (MPa)Shear Modulus (MPa)Shear Modulus (MPa)Airex C70.75600,9691,3450,8522Airex C70.75801,451042661,230

#### Table 2

#### Properties of ashwood skin material.

Material	Trademark	Density (kg/m <sup>3</sup> )	Compressive Strength, parallel to grain (MPa)	Flexural strength, longitudinal to grain direction (MPa)	Flexural Modulus (MPa)	Moisture content (%)
Ashwood	Novawood	650	69.3	90.7	13300	4–6

#### Table 3

Properties of rubber-cork intermediate sheet [38].

Material	Trademark	Density (kg/m <sup>3</sup> )	Tensile Strength (MPa)	Compressibility (%)	Hardness
Rubber-cork	TeknoCork	100–110	0.52	25–40	75 Shore A



Fig. 1. Schematic drawings of (a) wood skinned sandwich (SFS), (b) SF with rubber-cork intermediate layers (SFSR) and (c) multilayered wood skinned sandwich (MFSR) design concepts.

#### Table 4

In plane properties of E-glass/vinyl ester internal composite sheet.

Material properties Moduli	Values
Longitudinal Young's modulus (GPa)	20.3
Transverse Young's modulus (GPa)	19.2
Poisson's ratio	0.16
In-plane shear modulus (GPa)	3.4
Strengths	
Longitudinal tensile strength (MPa)	312
Longitudinal compressive strength (MPa)	112
Laminate shear strength (MPa)	43.3



Fig. 2. Schematic view of the impact test setup.

#### Table 5

Manufacturing matrix of specimens.

Structure	Specimen Code	Skin	Intermediate Layer	CORE			Intermediate Layer	Skin	Thickness (mm)	Specimen weight (gr)	Recyclable content (Vol. %)	Cost (€/m²)
SFS	H25	AW	_	_	HPVC25	_	_	AW	35	73	28.6	137
	H15	AW	_	_	HPVC15	_	_	AW	25	67	40	86
SFSR	H25R2	AW	R2	_	HPVC25	_	R2	AW	39	94	35.9	141
	H15R2	AW	R2	_	HPVC15	_	R2	AW	29	88	48.3	90
	H15R5	AW	R5	_	HPVC15	_	R5	AW	35	127	57.1	95
MFSR	HHR2	AW	R2	HPVC12	E-GLASS	HPVC12	R2	AW	39	134	35.9	157
	LLR2	AW	R2	LPVC12	E-GLASS	LPVC12	R2	AW	39	126	35.9	143
	LHR2	AW	R2	LPVC12	E-GLASS	HPVC12	R2	AW	39	130	35.9	150
	HLR2											

Description SFS: Single foam core sandwich design with ash wood skin, SFSR: Single foam core sandwich design with rubber-cork intermediate layer and ashwood skin, MFSR: Multi foam core sandwich design with E-glass sheet, rubber-cork intermediate layer and ashwood skin, AW: Ashwood, R2: 2 mm Rubber Cork, R5: 5 mm Rubber Cork, E-GLASS: 600 gr/m<sup>2</sup> bi-axial fabric

HPVC25: 25 mm PVC 80kg/m<sup>3</sup> AIREX C70.75 (Green), HPVC15: 15 mm PVC 80kg/m<sup>3</sup> AIREX C70.75 (Green)

**HPVC12:** 12 mm PVC 80kg/m<sup>3</sup> AIREX C70.75 (Green), **LPVC12:** 12 mm PVC 60kg/m<sup>3</sup> AIREX C70.55 (Yellow)

**Cost (€/m2):** It includes the total cost of all layers except ashwood skins.



Fig. 3. a) Load versus displacement curves and b) cross-sectional damage view of H25.



Fig. 4. a) Load versus displacement curves and b) cross-sectional damage view of H15.

this study is defined as the distance that impactor travels following the time of impact. Load-displacement curves comprising of ascending and descending sections can be divided into two main categories as open and closed curves [43–45]. Depending on the impact energy level, the descending section may represent rebounding, penetration and perforation cases. If the descending section is completely a softening curve, the load-displacement curve may be an open curve in that the impactor penetrates into the specimen or even perforates the specimen [45].

Figs. 3–11 illustrate the selected representative load-displacement graphs and cross-sectional images of impacted specimens for each sandwich configuration. Considering the thickness differences of the specimens, penetration depths and penetration depth to total thickness ratio of the specimens are given in Fig. 12(a–b).

The absorbed energy during an impact test was calculated by integrating the area under the load-displacement curve. This energy was absorbed by the sandwich specimen through the perforation of the skin as well as the crushing and failure process of the foam core [46]. The energy absorption is normalised by the total weight of the sandwich specimens to take into account the additional layers and the different densities of the foam core materials. Fig. 12(c) and (d) summarise the absorbed energy and the specific energy absorption of the sandwich specimens for complete perforation case. The maximum contact forces of all the sandwich specimens subjected to different impact energies are summarised in Fig. 12(e). The absorbed energy to cost ratio were calculated by dividing the absorbed energy to the unit areal cost of each configuration as seen in Fig. 12(f). The unit areal cost was calculated by summing up the cost of each individual constituent used per m<sup>2</sup> of the laminate. Since the geometry of the ashwood skin was kept constant in all sandwich specimens, its cost was not included into the calculation of unit areal cost.

#### 3.1. Single foam core sandwich (SFS) design with ashwood skin

Figs. 3(a) and 4 (a) show the load-displacement plots of H25 and H15 specimens, respectively. The impact response of SFS structure is a typical example of that associated with low-velocity impact on

sandwich structures, where two distinct peak points due to the upper and lower skin penetrations are clear in the load-displacement plots. In contrast to laminates, the graphs showing a mountain-like shape with two peaks are obtained when the penetration and/or perforation thresholds are exceeded in single-core sandwich structures [19,43,47]. Between these peaks (except for the 15 J where only the top skin was penetrated), there is a region where the force forms a plateau, which attributed to the displacement of the impactor through the foam core. The initial and secondary peak force values for these specimens were close due to perforation of the same material used for top and bottom skins.

Top ashwood skin failure and core cracking were observed at impact energy of 15 J in both specimens. The penetration depths were almost the same (Fig. 12a). However, the penetration depth to specimen thickness ratio was 48% in H15 specimen due to the lower thickness of the HPVC foam core while it was 34.3% for H25 specimen (Fig. 12b).

As the impact energy was increased to 30 J, the secondary peak was observed in the plots, indicating the partial perforation of the bottom ashwood skin of H25 and complete perforation of the H15 as seen in Figs. 3 and 4.

H25 specimens were completely perforated at the impact energy of 45 J. It can be seen from the load-displacement curves that H25 specimen exhibited a prolonged plateau region at impact energy of 45 J. The longer plateau results in higher energy absorption as seen in Fig. 12(c) and was related to the interaction of the impactor through the relatively thicker foam core in H25 compared to H15 specimens for the perforation case. Such results have been observed in previous studies [47,48]. The authors reported that with increasing the core thickness, the penetration can occur at a wider energy range under LVI loadings [48]. Also, it was observed that as the core thickness increased, specimens showed more elastic behaviour and so, the maximum contact force decreased while contact time and maximum deflection values were increasing. Comparing the samples H15 and H25 at 30 J energy level, the decrease in core thickness increased the maximum contact force as observed in Fig. 12e. Similar results were obtained with different core structures, for instance in sandwich containing balsa wood



Fig. 5. a) Load versus displacement curves and b) cross-sectional damage view of H25-R2.

core, the increase in core thickness causes an increase in the perforation threshold [47].

Although the H25 specimen was heavier, it absorbed higher specific energy than the H15 specimen. Similar results were observed in another study [49]. Authors found that the increase in the foam core thickness could improve the specific energy absorption amount of the same skinned sandwich composite under an impact event.

## 3.2. Single foam core sandwich design with rubber-cork intermediate layer and ashwood skin (SFSR)

Figs. 5–7 illustrate the load-displacement plots and cross-sectional images of SFSR specimens. At 15 J of impact energy, the rubber cork layer had an insignificant effect on the penetration depth. However, the deformation capability of the rubber cork layer prevented the impactor to reach the foam core, meaning a lesser damage to the core material. As illustrated in Fig. 12b, the penetration depth to thickness ratio was also slightly reduced from 34.3% to 30.8% with the introduction of the rubber cork layers in H25-R2 specimen. This reduction was from 48% to 41.4% in H15-R2 specimen. A comparison between the H15-R2 and H15-R5 specimens shows that the penetration depth to thickness ratio was reduced from 41.4% to 34.3% due to thicker rubber cork.

When the impact energy was increased to 30 J, SFSR specimens did not exhibit the second peak except the H15-R2 specimen. The penetration depth of H25-R2 and H15-R2 specimens was decreased in comparison to H25 and H15 specimens, respectively. Also, the penetration depth to thickness ratios were decreased about 20% compared to the SFS counterparts. Compared H25-R2 with H25, this reduction was observed from 97.1% to 76.9%. H15-R2 specimen showed a penetration depth to thickness ratio of 75.9% while H15 was completely perforated at 30 J energy level. H15-R5 specimens with the thicker rubber cork exhibited higher penetration depth in comparison to the H15-R2 specimens (see in Fig. 12a). The penetration depth to thickness ratio was almost the same due to increased thickness of the sandwich panel. This may be attributed to low stiffness of the rubber cork material.

All of the SFSR specimens were completely perforated at 45 J impact loading. The comparison of Figs. 4(a) and 6 (a) shows that H15-R2 specimen exhibits higher impact resistance than the H15 specimen. The same trend was also observed between the H25-R2 and H25 specimens. This suggests that the 2 mm thick rubber-cork layer increased the perforation threshold within the energy intervals used in the present study. Similarly, increasing the rubber-cork thickness increased the absorbed energy but decreased the specific absorbed energy due to the increased weight. Compared to the reference sample [48], the absorbed energy values in the SFSR specimens are comparable and higher but the SFS values are low. The improvement provided by the intermediate layer can be seen in Fig. 12c.

#### 3.3. Multi foam core sandwich design with E-glass internal sheet, rubbercork intermediate layer and ashwood skin (MFSR)

The load-displacement plots of MFSR specimens are given in Figs. 8–11. In order to evaluate the effect of alternative HPVC and LPVC foam core arrangements while the laminated composite sheet remains at the same location, tests were performed for four different design cases as tabulated in Table 5. These specimens were subjected to impact energies of 15, 45 and 75 J. As expected, the load- displacement curves are significantly different than those of SFS and SFSR designs due to the presence of the laminated composite sheet.

At 15 J of impact energy, the impactor perforated the ashwood skin in all four types of specimens with some differences. HH-R2 and HL-R2 type specimens, which have the HPVC at the impacted region, were subjected to core crushing while LL-R2 and LH-R2 specimens with the LPVC at the impacted region were suffered a significant core cracking. It can be said that the density of the foam material in the direction of impact at low energy levels affects the damage state. At the 15 J energy level, HH-R2 and HL-R2 showed a penetration depth to thickness ratio of 30.8% while it was 35.9% for LL-R2 and LH-R2 specimens (see in Fig. 12b).



Fig. 6. a) Load versus displacement curves and b) cross-sectional damage view of H15-R2.



Fig. 7. a) Load versus displacement curves and b) cross-sectional damage view of H15-R5.



Fig. 8. a) Load versus displacement curves and b) cross-sectional damage view of HH-R2.



Fig. 9. a) Load versus displacement curves and b) cross-sectional damage view of LL-R2.



Fig. 10. a) Load versus displacement curves and b) cross-sectional damage view of HL-R2.

In the case of 45 J of impact energy, the impactor contacts with the E-glass internal sheet and reveals a force peak in the load-displacement curves (see Figs. 8–11). This interaction caused a fibre breakage and inter-laminar delamination in the E-glass sheet. Although the impactor did not get into contact with the lower core, it was crushed due to the

load transferred through the internal sheet. The delamination width were determined by measuring over the damaged specimens by a vernier caliper. It is also seen from the Figs. 8 and 9 that the extent of delamination are was reduced by introducing HPVC foam core foam on the sides of the internal sheet (Fig. 8b). The higher deformation



Fig. 11. a) Load versus displacement curves and b) cross-sectional damage view of LH-R2.



Fig. 12. LVI test results; a) penetration depth, b) penetration depth/specimen thickness, c) absorbed energy, d) specific absorbed energy, e) maximum contact force and f) absorbed energy/cost.

capability of LPVC foam caused a larger delamination area due to stiffness incompatibility (Fig. 9b). At 45 J energy level, the delamination width at LL-R2 specimen was about 60 mm while it was around 40 mm at HH-R2 specimen. In the HL-R2 and LH-R2 specimens, the internal sheet was broken at this energy level (Figs. 10b and 11 b). At this energy level, the LL-R2 samples showed a 69.2% penetration depth to thickness value, while the others remained at the same level of 64.1% (see, Fig. 12b).

For the impact energy of 75 J, all curves are in an open form corresponding to a full perforation case in which all the ashwood skins, internal composite sheet and the core material were perforated. At this energy level, it is clear that the first, second and third peaks represent failure in the upper ashwood skin, internal composite sheet and lower ashwood skin, respectively. LL-R2 specimens experienced the lowest energy absorption because of the low density PVC core material. In contrast, HH-R2 comprising of the high density PVC resulted highest absorbed energy (see Fig. 12 (c)). This confirms the earlier result, which indicates that stiffer foam core material in the entire thickness of the core reduces the impact induced damage of multilayered sandwich panels [25,26]. Compared the absorbed energy values of HL-R2 and LH-

R2, placing the higher density foam uppermost resulted in a slight increase in the perforation resistance (see in Figs. 12c). Such results were observed in the impact response of the graded foam sandwich constructions [50]. The cross-sectional views of the MFSR specimens revealed that using a high density PVC foam core created a more impact-resistant structure under LVI loading.

#### 4. Conclusions

Industrial woods inherently fail in a brittle manner which makes them susceptible to impact loadings. Despite this weakness, they are considered as eco-friendly materials due to their recyclability and alternative structural configurations have been evaluated in order to increase their impact resistance.

In the present paper, innovative, eco-friendly and impact resistant sandwich structure designs were developed using recyclable and longlasting thermowood and rubber-cork materials. Three different concepts were presented in order to evaluate their performance under low velocity impact loading. Cross-sectional images of the damaged specimens enabled visual inspection of interior damage patterns and penetration depth values. As a result, this research shed light on the LVI behavior of environmentally friendly wood-skinned sandwich composites. The results can be used to improve the LVI damage tolerance of commercial available PVC core wood marine sandwich panels. Fundamental findings are summarised below:

- The absorbed energy values were obtained from the test data and represent the area under the load-displacement graphs. For the 15 J of impact energy, the upper skin was perforated and absorbed 34.8% of the impact energy while 59.2% of that was absorbed by the foam core and/or the rubber cork damage depending on the structure. At 12 mm final penetration, 94% of impact energy was absorbed and 6% was spent as rebounding case.
- Introduction of the rubber cork intermediate layer resulted in reduced penetration depth for 30 J of impact energy by increasing the energy absorption. At low level of impact energy, the rubber cork prevented foam cracking, and the damage was therefore confined to the recyclable materials only. This is an advantage for the repair processes and to avoid the progressive cracking of the foam upon further loading cycles.
- Penetration depth was increased with increasing the rubber cork thickness due to the replacement of stiff foam core by a relatively flexible rubber cork material on the travel path of the impactor. However, the absorbed energy was increased about 10% while the specific absorbed energy decreased due to weight gain of the sandwich structure.
- The maximum contact force values were between 1.5 and 2 kN for SFS and SFSR structures. In MFSR structures it was raised up to 4.5 kN due to introduction of high stiffness E-glass internal composite layer. The higher contact force was resulted in higher perforation threshold and absorbed energy. For the non-perforating energy levels, high density foam core at the impacted side of the MFSR design may be used to considerably lighten the structure.
- From the economical perspective of view, with respect to the specific absorbed energy and absorbed energy/cost ratios, HH-R2 was found to be the best panel configuration among the proposed design alternatives.
- The introduction of the rubber-cork material increased the recyclability rate of the sandwich structures up to 57%. In the MFSR configurations, the recyclability rate is 35.6%.

The proposed wood skinned sandwich composite designs are easy to manufacture, repair and recycle in addition to their valuable esthetical appearance. These configurations may be utilised in floor applications such as decks and cabins, bulkhead panels as well as interior and exterior design applications.

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